

Networked and Distributed Parameter Systems

(Some) New Directions, Opportunities & Challenges

Bassam Bamieh

Mechanical Engineering

Center for Control, Dynamical Systems and Computation

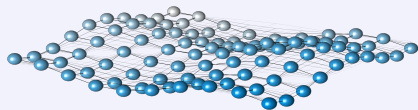
UNIVERSITY OF CALIFORNIA AT SANTA BARBARA



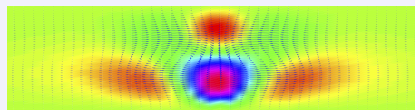
Networked vs. Distributed Parameter Systems

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



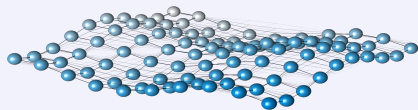
Distributed Parameter Systems



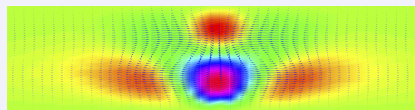
Networked vs. Distributed Parameter Systems

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control

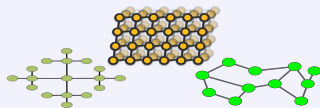


Distributed Parameter Systems

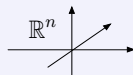


CORRESPONDENCES (Physics/Numerical Analysis perspective)

discrete space described by graph structure



continuum space



*differential equations
over large graphs*

Numerical Methods



Continuum Models

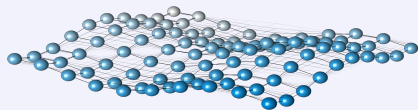


*Partial Differential
Equations*

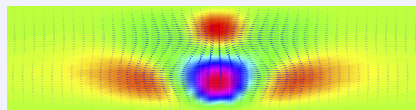
Networked vs. Distributed Parameter Systems

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control

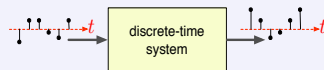
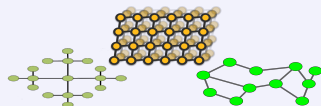


Distributed Parameter Systems

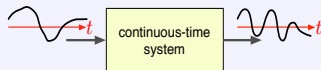
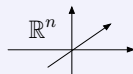


ANALOGY WITH TEMPORAL SYSTEMS (Systems & Controls perspective)

discrete space described by graph structure



continuum space



UNIFYING PERSPECTIVE:

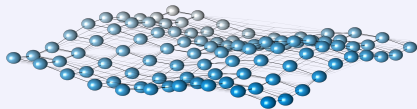
Spatio-temporal systems over discrete or continuum space

- Signals over continuous and/or discrete time and space
- Investigate systems properties (e.g. system norms & responses)

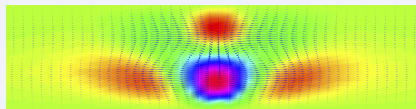
Outline

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



LOOK AT SPECIFIC PROBLEMS

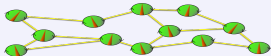
— Vehicular Strings and Consensus



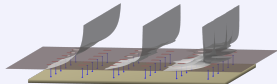
— Structured Control Design



— Synchrony in AC Power Networks



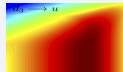
— Flow Turbulence & Control



— Spatio-temporal



Impulse Responses

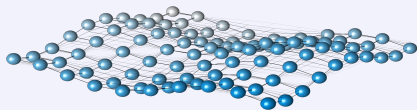


Frequency Responses

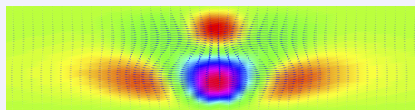
Outline

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



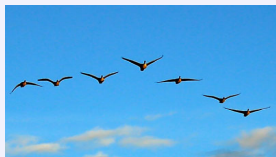
SOME COMMON THEMES EMERGE

- *The use of system norms and responses*
- *Large-scale (even linear) systems exhibit some surprising phenomena*
- *Large-scale & Regular Networks* → *Asymptotic statements (in system size)*
- *Network topology imposes asymptotic “hard performance limits”*

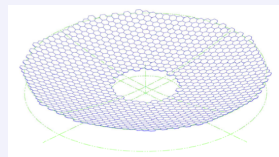
Networked/Cooperative/Distributed Control



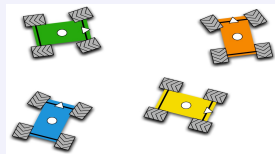
aircraft formation flight



formation flight in nature



large telescope arrays



robotic networks



flocks & swarms

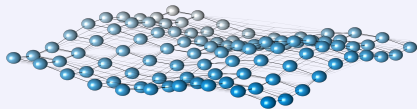


automated highways

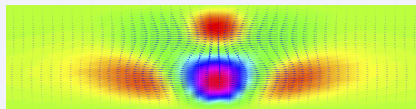
VEHICULAR STRINGS (PLATOONS)

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



LOOK AT SPECIFIC PROBLEMS

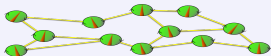
— Vehicular Strings and Consensus



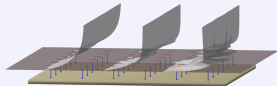
— Structured Control Design



— Synchrony in AC Power Networks



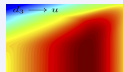
— Flow Turbulence & Control



— Spatio-temporal



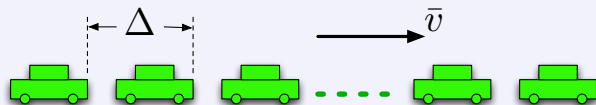
Impulse Responses



Frequency Responses

Vehicular Platoons

Automated control of each vehicle, tight spacing at highway speeds

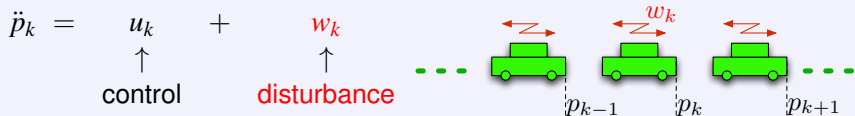


- Is it enough to look at neighbors? Should information be broadcast to all?
- How does performance scale with size?
- Are there any fundamental limitations?

A fundamentally difficult problem (scales badly with size)

due to the network topology

Vehicular Platoons (setting)



- *Desired trajectory*: $\bar{p}_k := \bar{v}t + k\Delta$ *constant velocity*

- Deviations:

$$\tilde{p}_k := p_k - \bar{p}_k, \quad \tilde{v}_k := \dot{p}_k - \bar{v}$$

- Controls:

$$u = K\tilde{p} + F\tilde{v}$$

- Closed loop:

$$\frac{d}{dt} \begin{bmatrix} \tilde{p} \\ \tilde{v} \end{bmatrix} = \begin{bmatrix} 0 & I \\ K & F \end{bmatrix} \begin{bmatrix} \tilde{p} \\ \tilde{v} \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} w$$

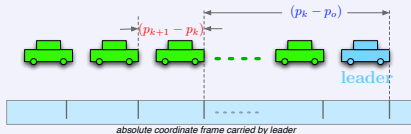
K, F : matrix feedback gains (look like “Laplacians” \approx 2nd order consensus)

Relative vs. Absolute Feedback

$$u = \begin{array}{c} \text{position feedback} \\ \downarrow \\ K \tilde{p} \end{array} + \begin{array}{c} \text{velocity feedback} \\ \downarrow \\ F \tilde{v} \end{array}$$

$$u_k = K_+ (p_{k+1} - p_k - \Delta) + F_+ (v_{k+1} - v_k)$$

$$K_o (p_k - (vt + \Delta k)) + F_o (v_k - \bar{v})$$



$$+ K_- (p_k - p_{k-1} - \Delta) + F_- (v_k - v_{k-1})$$

+

- **RELATIVE MEASUREMENTS:**

- ▶ Requires ranging devices

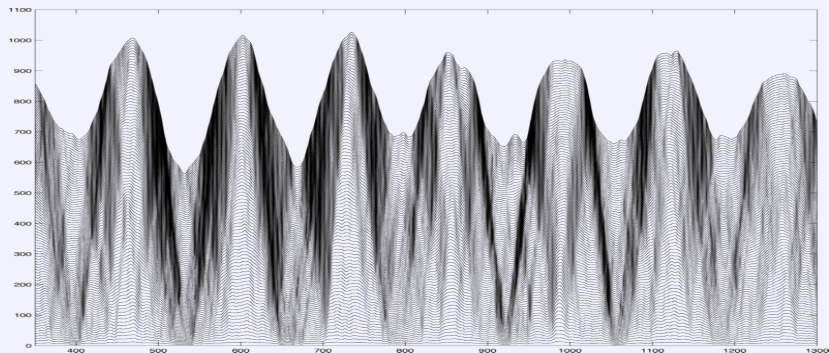
$$\Leftrightarrow \begin{array}{l} \text{row_sums}(K) = 0 \\ \text{row_sums}(F) = 0 \end{array}$$

- **ABSOLUTE MEASUREMENTS:**

- ▶ Position: Requires knowing position relative to leader
- ▶ Velocity: Requires measurement of own velocity

Disorder Phenomenon in Platoons (w. only relative meas.)

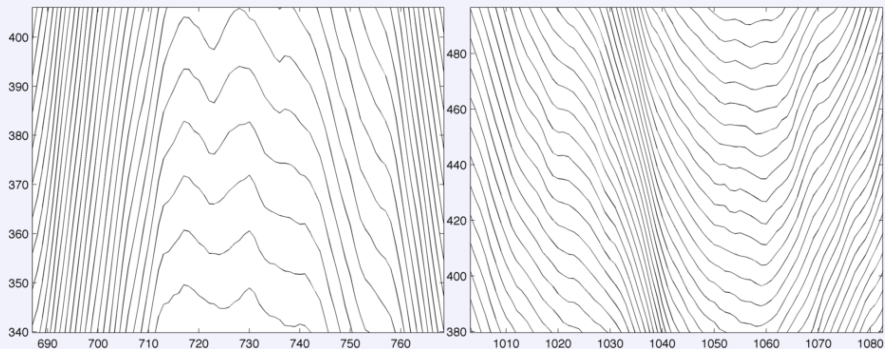
Globally stable formation, but exhibits “accordion-like” large-scale modes



Time trajectories of vehicles' positions relative to leader (bird's-eye view)
100 vehicles

-A large formation in a **thunderstorm**

Zoomed in (small-scale) behavior

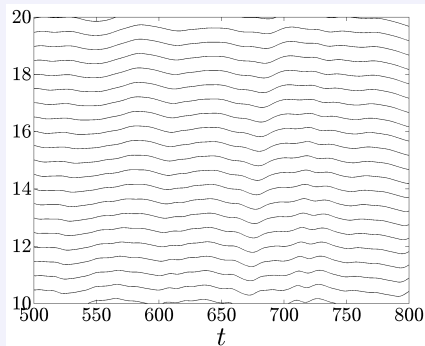
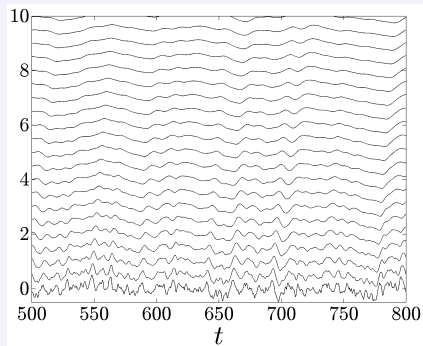


Seems well regulated. No collisions.

Disorder Phenomenon in Platoons (w. only relative meas.)

String instability?

Let disturbances enter only at lead vehicle

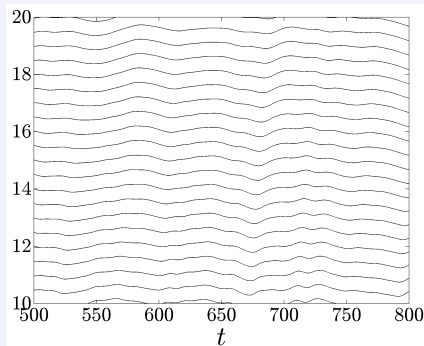
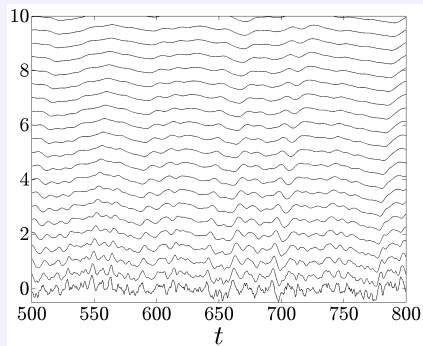


Unrelated to string instability!

Disorder Phenomenon in Platoons (w. only relative meas.)

String instability?

Let disturbances enter only at lead vehicle



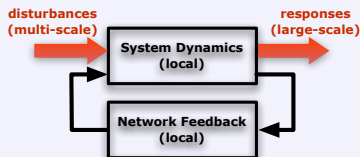
- temporally high frequency disturbances well regulated
- temporally low frequency disturbances penetrate further into formation

Vehicular Platoons (Optimal LQR)

- Is this due to bad design, or is it inherent to this problem?
- Note: Also occurs in LQR controllers that yield “localized” feedbacks
 - ▶ Original formulations:
 - ★ Athans & Levine '66
 - ★ Melzer & Kuo '70
 - ▶ Reexamined as $N \rightarrow \infty$
 - ★ Jovanovic & Bamieh, *TAC* '05

Disorder and Feedback “Granularity”

- Disturbances are spatially white (contain all spatial wavelengths)

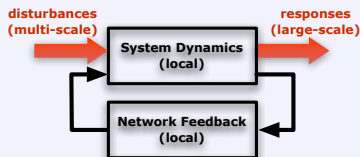


- Intuition:

- ▶ *Local feedback* can only suppress *short-scale disturbances*
- ▶ Local feedback ineffective against
large-scale (& slow) disturbances
- ▶ Looks like *global feedback* is needed for *global regulation*

Disorder and Feedback “Granularity”

- Disturbances are spatially white (contain all spatial wavelengths)



- Intuition:

- ▶ *Local feedback* can only suppress *short-scale disturbances*
- ▶ Local feedback ineffective against large-scale (& slow) disturbances
- ▶ Looks like *global feedback* is needed for *global regulation*

Surprise: In higher spatial dimensions:

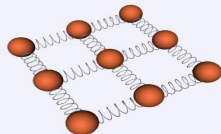
Local feedback CAN suppress large-scale disturbances
cf. Harmonic Solids

Statistical Mechanics of Harmonic Solids

Harmonic solid: A d -dimensional lattice of masses and springs

Q: *Can short range interaction lead to long range order?*

- “short range interaction” \longleftrightarrow local feedback
- “long range order” \longleftrightarrow tightness of formation

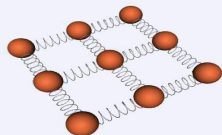


Statistical Mechanics of Harmonic Solids

Harmonic solid: A d -dimensional lattice of masses and springs

Q: *Can short range interaction lead to long range order?*

- “short range interaction” \longleftrightarrow local feedback
- “long range order” \longleftrightarrow tightness of formation



Studied using *long range correlations*

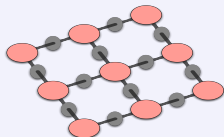
- for $d = 1, 2$ short range interactions \Rightarrow no long range order
- for $d \geq 3$ long range order possible!
- i.e., solids can only exist in $d \geq 3$

Statistical Mechanics of Harmonic Solids

Harmonic solid: A d -dimensional lattice of masses and springs

Q: *Can short range interaction lead to long range order?*

- “short range interaction” \longleftrightarrow local feedback
- “long range order” \longleftrightarrow tightness of formation



Studied using *long range correlations*

- for $d = 1, 2$ short range interactions \Rightarrow no long range order
- for $d \geq 3$ long range order possible!
- i.e., solids can only exist in $d \geq 3$

Similar dimensional-dependencies occur in networked control systems?

Related Concepts

- Optimal Performance of Distributed Estimation [Barooah, Hespanha]
- Effective Resistance in a Resistor Network
- Global Mean First Passage Time of Simple Random Walk
- Wiener Index for Molecules

Common mathematical problem: calculate sums like (cont. time)

$$\sum_{n \neq 1} \frac{1}{\lambda_n}$$

λ_n : eigenvalues of a graph Laplacian

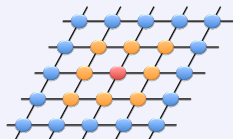
Setting:

- $N = M^d$ vehicles arranged in d-dimensional torus \mathbb{Z}_M^d
- Desired trajectory: $\bar{p}_k := vt + k\Delta$ constant speed & heading

Structural Constraints

- *Spatial Invariance:*
State-feedbacks K and F are spatial-convolution operators
- *Locality:* $K_{(k_1, \dots, k_d)} = 0$, if for any $i \in \{1, \dots, d\}$, $|k_i| > q$

feedback from *local neighbors only*

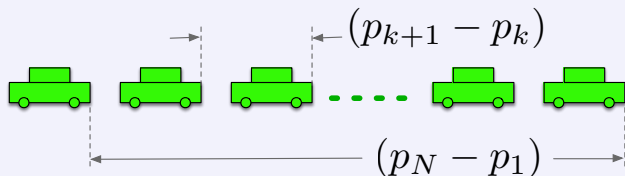


Performance Measures

- Two measures of “disorder”

- ▶ **Microscopic:** *local position deviation*

$$\text{var}(p_{k+1} - p_k - \Delta)$$



- ▶ **Macroscopic:** *long range deviation*

$$\text{var}(p_N - p_1 - \Delta N)$$

or

$$\text{var}\left(\tilde{p}_k - \frac{1}{N} \sum_l \tilde{p}_l\right)$$

- All above obtained *asymptotically* (as $N \rightarrow \infty$) from H^2 norm calculations

Asymptotic Performance Lower Bounds

Tori networks, network size = N , spatial dimension = d , control effort = $\mathcal{E}\{u_k^2\} \leq U$

Feedback Type	Microscopic Disorder	Macroscopic Disorder
1st order consensus	$\frac{1}{U}$	$\frac{1}{U} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$
absolute position & absolute velocity	$\frac{1}{U}$	$\frac{1}{U}$
relative position & absolute velocity	$\frac{1}{U}$	$\frac{1}{U} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$
relative position & relative velocity	$\frac{1}{U^2} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$	$\frac{1}{U^2} \begin{cases} N^3 & d = 1 \\ N & d = 2 \\ N^{1/3} & d = 3 \\ \log(N) & d = 4 \\ 1 & d \geq 5 \end{cases}$

“Coherence in Large-Scale Networks: Dimension-Dependent Limitations of Local Feedback”

BB, Jovanovic, Mitra, Patterson TAC, 2012

Implications for Vehicular Platoons

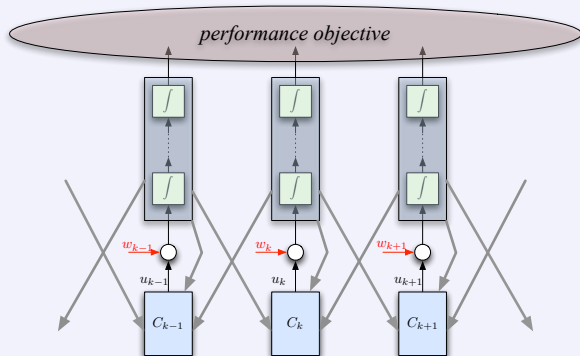
Feedback Type	Microscopic Disorder	Macroscopic Disorder
1st order consensus	$\frac{1}{W}$	$\frac{1}{W} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$
absolute position & absolute velocity	$\frac{1}{W}$	$\frac{1}{W}$
relative position & absolute velocity	$\frac{1}{W}$	$\frac{1}{W} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$
relative position & relative velocity	$\frac{1}{W^2} \begin{cases} N & d = 1 \\ \log(N) & d = 2 \\ 1 & d \geq 3 \end{cases}$	$\frac{1}{W^2} \begin{cases} N^3 & d = 1 \\ N & d = 2 \\ N^{1/3} & d = 3 \\ \log(N) & d = 4 \\ 1 & d \geq 5 \end{cases}$

Using only local feedback:

cannot have 1 dimensional, large and yet coherent formations!


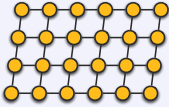
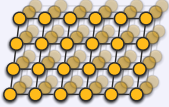
Role of Node Dynamics

- Each node a chain of n integrators
- Controllers use local *static* state feedback



- Critical dimension needed for global coherence = $2n + 1$
- Tradeoff between *network connectivity* and *node memory*


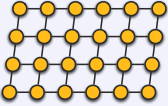
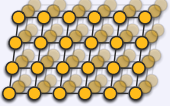
Spatial Dimension and Network Connectivity

dimension	 $d = 1$	 $d = 2$	 $d = 3$	d-dimensional Torus (Lattice) ($d \geq 4$)
macroscopic disorder $\frac{1}{N} \sum_{n \neq 1} 1/\lambda_n$ 1 st -order consensus	N	$\log(N)$	bounded	bounded

- Node degree does not quantify this phenomenon

- e.g. compare  with 

Spatial Dimension and Network Connectivity

dimension	 $d = 1$	 $d = 2$	 $d = 3$	d-dimensional Torus (Lattice) ($d \geq 4$)
macroscopic disorder $\frac{1}{N} \sum_{n \neq 1} 1/\lambda_n$ 1 st -order consensus	N	$\log(N)$	bounded	bounded

- Node degree does not quantify this phenomenon

- e.g. compare  with 

- Note:** $\frac{1}{N} \sum_{n \neq 1} 1/\lambda_n$ scales differently from $1/\lambda_2$

Coherence Analysis in General Graphs?

For general graphs, what is the corresponding notion of “spatial dimension”?



(opte.org)

- The *Hausdorff dimension* of a fractal graph does not fully characterize coherence
Patterson, BB, '11 CDC
- Open question: a *purely topological* measure of coherence for general graphs

Swarms and Flocks in Nature

1d



2d



3d



- Network dimensionality determines *coherence of motion*?

Starling Flocks: Young, Scardovi, Cavagna, Giardina, Leonard, '13, PLOS CB

Further Questions

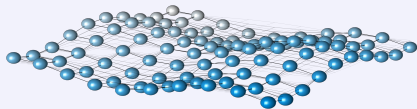
- Can more general control laws break these limitations?
 - ▶ Spatial varying control gains?
 - ▶ Nonlinear feedback?
 - ▶ Dynamic feedback?

- Must have global feedback to address coherence problem
 - ▶ *Vulnerability to errors in global feedback (as $N \rightarrow \infty$)?*

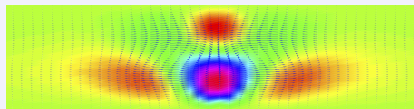
STRUCTURED, DISTRIBUTED CONTROL DESIGN

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems

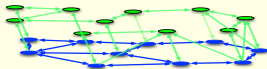


LOOK AT SPECIFIC PROBLEMS

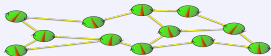
— Vehicular Strings and Consensus



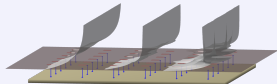
— Structured Control Design



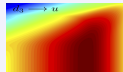
— Synchrony in AC Power Networks



— Flow Turbulence & Control



— Spatio-temporal

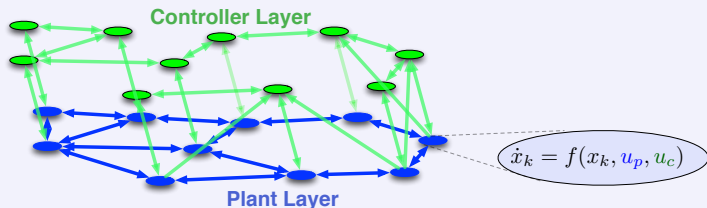


Impulse Responses

Frequency Responses

Distributed Control Systems Design

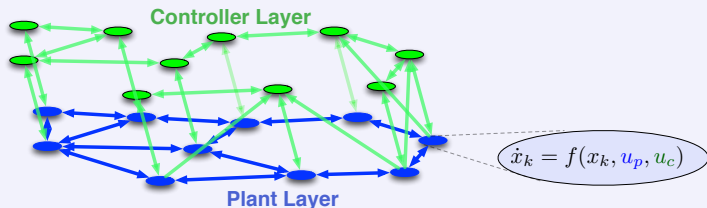
- **Controller Architecture:** Constraints on controller information flow



- **Optimal Constrained Controller Design**
 - ▶ In general: difficult, non-convex, non-scalable
 - ▶ Some Exceptions:
 - ★ Partially Nested Info. Structure, Funnel Causality, Quadratic Invariance
 - ★ Sparsity Promoting (ℓ^1 -regularized) designs
 - ▶ Often possible to propose (non-optimal), scalable algorithms that “work”
 - ★ e.g. Consensus-like algorithms (cf. multi-agent systems)

Distributed Control Systems Design

- **Controller Architecture:** Constraints on controller information flow



- **Optimal Constrained Controller Design**
 - ▶ In general: difficult, non-convex, non-scalable
 - ▶ Some Exceptions:
 - ★ Partially Nested Info. Structure, Funnel Causality, Quadratic Invariance
 - ★ Sparsity Promoting (ℓ^1 -regularized) designs
 - ▶ Often possible to propose (non-optimal), scalable algorithms that “work”
 - ★ e.g. Consensus-like algorithms (cf. multi-agent systems)
- **Q: Why care about optimality?**

Quantify fundamental limitations-of-performance due to *network topology*?
akin to those due to RHP poles/zeros

Why care about difficult *optimal/robust* control problems?

- **Optimality** gives *Best Achievable Limits of performance*

- ▶ e.g. a plant G with a RHP pole p and zero z

$$\inf_{C \text{ stabilizing}} \|(1 + PC)^{-1}\|_{\infty} = \frac{|z + p|}{|z - p|} \checkmark$$

- ▶ If $z \neq p$, system is both controllable/observable, the rank tests

$$\text{rank} [B \ AB \ \cdots \ A^{n-1}B] \quad \text{rank} [C; \ CA; \ \cdots ; \ CA^{n-1}]$$

give a deceptive answer!

(especially for large-scale systems!)

Why care about difficult *optimal/robust* control problems?

- **Optimality** gives *Best Achievable Limits of performance*

- ▶ e.g. a plant G with a RHP pole p and zero z

$$\inf_{C \text{ stabilizing}} \|(1 + PC)^{-1}\|_{\infty} = \frac{|z + p|}{|z - p|} \checkmark$$

- ▶ If $z \neq p$, system is both controllable/observable, the rank tests

$$\text{rank} [B \ AB \ \cdots \ A^{n-1}B] \quad \text{rank} [C; CA; \cdots ; CA^{n-1}]$$

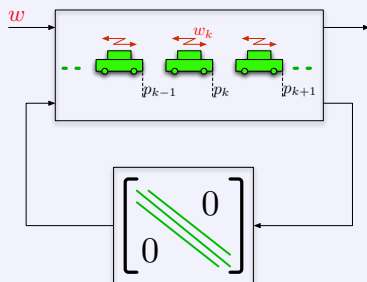
give a deceptive answer! (especially for large-scale systems!)

- ▶ Use $\inf_{C \text{ stabilizing}} \|\mathcal{F}(G; C)\|$ to measure
approximate network controllability/observability
- ▶ Optimal/Robust Control is useful to
design/characterize a good plant, not just controller!

A point recognized in 80's-90's, but has not made it into networks literature

Case Study: Vehicular Formations

Vehicular string control with only local (no leader) information



- Corresponds to banded controller structure
- This exact problem is non-convex (currently unsolved)
- *Can find lower bounds (hard performance limits) as function of topology!*
- *The platoons problem is fundamentally difficult because of the 1d topology*

Structured Optimal Control in the Limit of Large System Size

- The problem $\inf_{C \text{ structured}} \|\mathcal{F}(G; C)\|$
 - ▶ very difficult for finite N
 - ▶ may admit simple answers as $N \rightarrow \infty$
 - ▶ cf. *Statistical Mechanics*

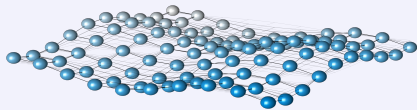


- Use structured Robust/Optimal control problems
not to design network controllers, but to quantify *limits of performance*
- Implications:
 - ▶ In engineered systems: allows for selection of network structures
 - ▶ In natural systems (e.g. biological): may explain naturally evolved network structures
 - ▶ *Quantify network controllability/observability*

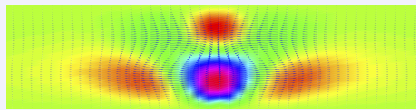
SYNCHRONY IN AC POWER NETWORKS

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



LOOK AT SPECIFIC PROBLEMS

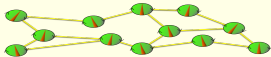
— Vehicular Strings and Consensus



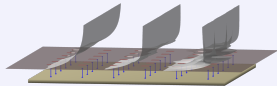
— Structured Control Design



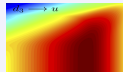
— Synchrony in AC Power Networks



— Flow Turbulence & Control



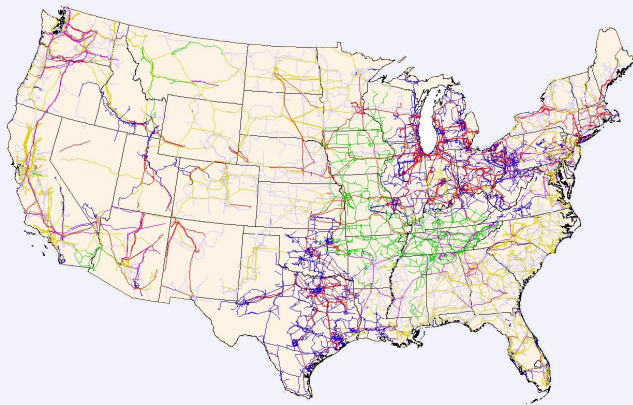
— Spatio-temporal



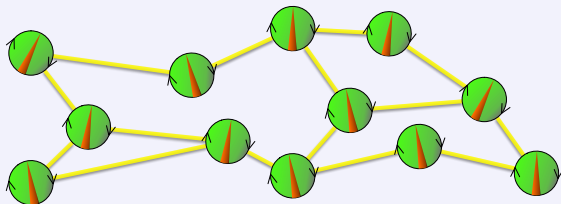
Impulse Responses

Frequency Responses

AC Power Networks



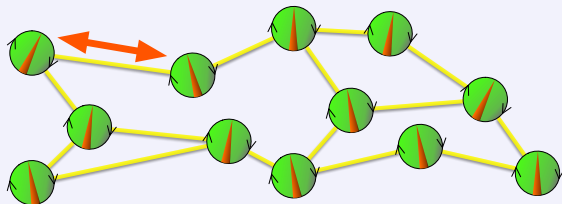
Phase Synchronization in AC Networks



- Machines “tug” on each other to achieve phase synchrony
Linearized dynamics (swing equations) similar to vehicle formations

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \omega \end{bmatrix} = \begin{bmatrix} 0 & I \\ -L_B & -\beta I \end{bmatrix} \begin{bmatrix} \theta \\ \omega \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} w$$

Phase Synchronization in AC Networks



- Machines “tug” on each other to achieve phase synchrony
Linearized dynamics (swing equations) similar to vehicle formations

$$\frac{d}{dt} \begin{bmatrix} \theta \\ \omega \end{bmatrix} = \begin{bmatrix} 0 & I \\ -L_B & -\beta I \end{bmatrix} \begin{bmatrix} \theta \\ \omega \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} w$$

- **Electrical power** flows back and forth as a **signaling mechanism**

A Thought Experiment: Network with Identical Generators

- Assume *identical generators* but *general topology*

$$\begin{aligned} \frac{d}{dt} \begin{bmatrix} \theta \\ \omega \end{bmatrix} &= \begin{bmatrix} 0 & I \\ -L_B & -\beta I \end{bmatrix} \begin{bmatrix} \theta \\ \omega \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} w \\ y &= \begin{bmatrix} C_1 & 0 \end{bmatrix} \end{aligned}$$

- Resistive power loss over (i, k) link

$$\tilde{P}_{loss_{ik}} = g_{ik} |\theta_i - \theta_k|^2$$

- Total resistive losses $\tilde{\mathbf{P}}_{loss} = y^* y$

$$C_1^* C_1 := L_G,$$

- Notes

- ▶ Network Admittance Matrix: $Y = \text{Re}\{Y\} + j\text{Im}\{Y\} =: L_G + jL_B$
- ▶ Linearized dynamics
- ▶ Keep only quadratic part of loss term

- ▶ **Model too simple?**

Note: Modeling *best case scenario*, **no instabilities**

Calculating the H^2 Norm

Assumption: L_G is a multiple of L_B

$$\alpha := \frac{g_{ik}}{b_{ik}} = \frac{r_{ik}}{x_{ik}} = \text{ratio of line resistance to reactance}$$

Then *total resistive power loss*

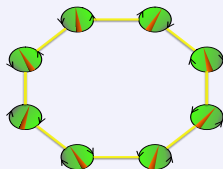
$$E\{y^*y\} = \frac{\alpha}{\beta} (N - 1)$$

N : Network Size

Total resistive losses are Independent of the network topology!!

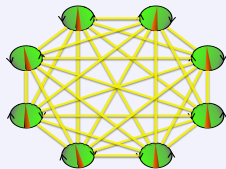
Implications

Compare:



less coherent
larger phase fluctuations
less links
Resistive losses

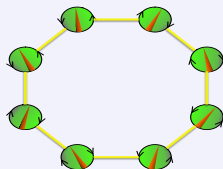
vs.



<
>
<
=
more coherent
small phase fluctuations
more links
Resistive losses

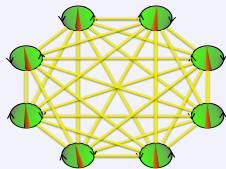
Implications

Compare:



less coherent
larger phase fluctuations
less links
Resistive losses

vs.



< more coherent
> small phase fluctuations
< more links
= Resistive losses

- A fundamental limitation, independent of network topology
A consequence of using *electrical power flows* as the signaling mechanism!

"The Price of Synchrony", BB, Gayme, '13, ACC

- Losses proportional to network size N

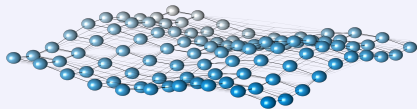
What if $N \approx$ millions in a future highly-distributed-generation smart grid??

Another argument for a communications layer in the smart grid

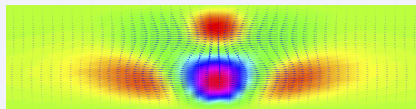
FLOW TURBULENCE & CONTROL

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



LOOK AT SPECIFIC PROBLEMS

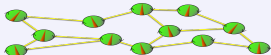
— Vehicular Strings and Consensus



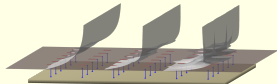
— Structured Control Design



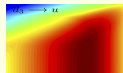
— Synchrony in AC Power Networks



— Flow Turbulence & Control



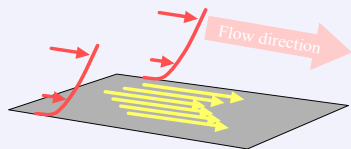
— Spatio-temporal



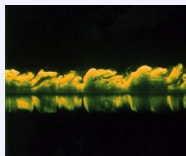
Impulse Responses

Frequency Responses

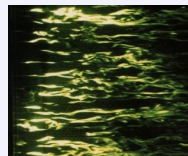
Turbulence in Streamlined Flows (Boundary Layers)



boundary layer turbulence

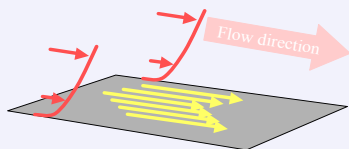


side view

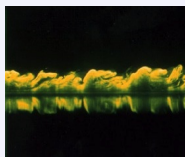


top view

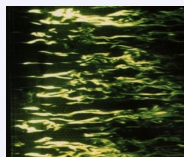
Turbulence in Streamlined Flows (Boundary Layers)



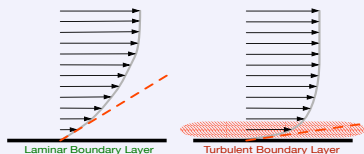
boundary layer turbulence



side view



top view

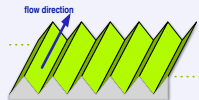
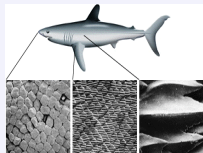


skin-friction drag: laminar vs. turbulent

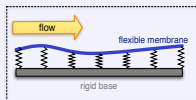
- Streamlining a vehicle reduces *form drag*
- Still stuck with: **Skin-Friction Drag** (higher in *Turbulent BL* than in *Laminar BL*)
- Same in pipe flows (*increases required pumping power*)

Control of Boundary Layer Turbulence

in nature: "passive" control

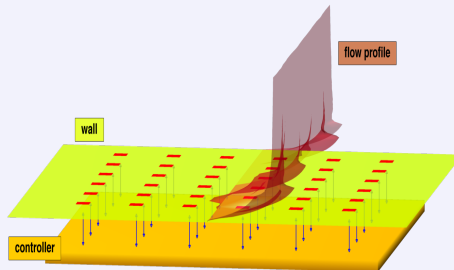


corrugated skin



compliant skin

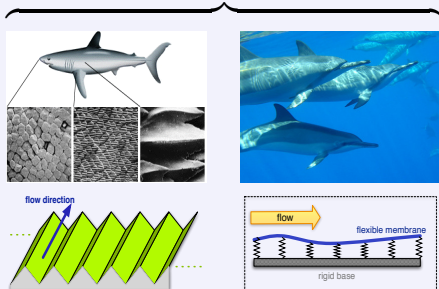
active control with
sensor/actuator arrays



- Intuition: must have ability to actuate at spatial scale comparable to flow structures
spatial-bandwidth of controller \geq plant's bandwidth

Control of Boundary Layer Turbulence

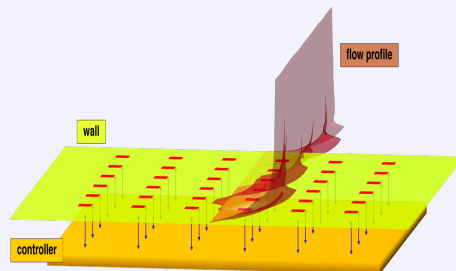
in nature: “passive” control



corrugated skin

compliant skin

active control with
sensor/actuator arrays

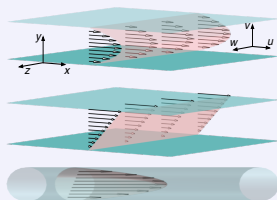


- Intuition: must have ability to actuate at spatial scale comparable to flow structures
spatial-bandwidth of controller \geq plant's bandwidth
- **Caveat:** *Plant's dynamics are not well understood*
obstacles $\left\{ \begin{array}{l} \text{not only device technology} \\ \text{also: dynamical modeling and control design} \end{array} \right.$

Mathematical Modeling of Transition: Hydrodynamic Stability

The Navier-Stokes (NS) equations:

$$\begin{aligned}\partial_t \mathbf{u} &= -\nabla_{\mathbf{u}} \mathbf{u} - \text{grad } p + \frac{1}{R} \Delta \mathbf{u} \\ 0 &= \text{div } \mathbf{u}\end{aligned}$$

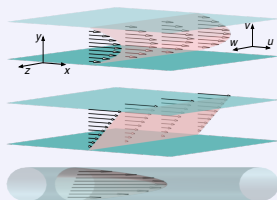


- Hydrodynamic Stability: view NS as a dynamical system
- *laminar flow* $\bar{\mathbf{u}}_R$:= a stationary solution of the NS equations (an *equilibrium*)

Mathematical Modeling of Transition: Hydrodynamic Stability

The Navier-Stokes (NS) equations:

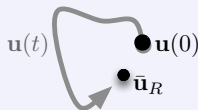
$$\begin{aligned}\partial_t \mathbf{u} &= -\nabla_{\mathbf{u}} \mathbf{u} - \text{grad } p + \frac{1}{R} \Delta \mathbf{u} \\ 0 &= \text{div } \mathbf{u}\end{aligned}$$



- Hydrodynamic Stability: view NS as a dynamical system
- *laminar flow* $\bar{\mathbf{u}}_R$:= a stationary solution of the NS equations (an *equilibrium*)

laminar flow $\bar{\mathbf{u}}_R$ stable \longleftrightarrow i.c. $\mathbf{u}(0) \neq \bar{\mathbf{u}}_R$,
 $\mathbf{u}(t) \xrightarrow{t \rightarrow \infty} \bar{\mathbf{u}}_R$

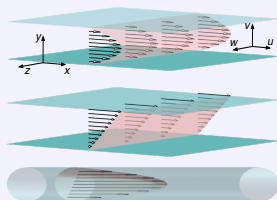
- ▶ typically done with dynamics linearized about $\bar{\mathbf{u}}_R$
- ▶ various methods to track further “non-linear behavior”



Mathematical Modeling of Transition: Hydrodynamic Stability

The Navier-Stokes (NS) equations:

$$\begin{aligned}\partial_t \mathbf{u} &= -\nabla_{\mathbf{u}} \mathbf{u} - \text{grad } p + \frac{1}{R} \Delta \mathbf{u} \\ 0 &= \text{div } \mathbf{u}\end{aligned}$$



- Hydrodynamic Stability: view NS as a dynamical system
- A very successful (*phenomenologically predictive*) approach for many decades
- **However:** *it fails badly in the special (but important) case of streamlined flows*

Mathematical Modeling of Transition: Adding Signal Uncertainty

- Decompose the fields as

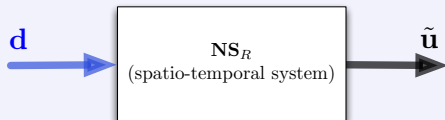
$$\mathbf{u} = \begin{array}{c} \bar{\mathbf{u}}_R \\ \uparrow \\ \text{laminar} \end{array} + \begin{array}{c} \tilde{\mathbf{u}} \\ \uparrow \\ \text{fluctuations} \end{array}$$

- Fluctuation dynamics:

In *linear* hydrodynamic stability, $-\nabla_{\tilde{\mathbf{u}}}\tilde{\mathbf{u}}$ is ignored

$$\begin{aligned} \partial_t \tilde{\mathbf{u}} &= -\nabla_{\bar{\mathbf{u}}_R} \tilde{\mathbf{u}} - \nabla_{\tilde{\mathbf{u}}} \bar{\mathbf{u}}_R - \text{grad } \tilde{p} + \frac{1}{R} \Delta \tilde{\mathbf{u}} - \nabla_{\tilde{\mathbf{u}}} \tilde{\mathbf{u}} + \mathbf{d} \\ 0 &= \text{div } \tilde{\mathbf{u}} \end{aligned}$$

- a time-varying *exogenous disturbance* field \mathbf{d}



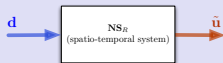
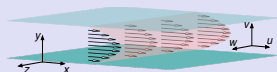
Input-Output view of the Linearized NS Equations

Jovanovic, BB, '05 JFM

Input-Output Analysis of the Linearized NS Equations

$$\partial_t \begin{bmatrix} \Delta \tilde{v} \\ \tilde{\omega} \end{bmatrix} = \begin{bmatrix} U'' \partial_x - U \Delta \partial_x + \frac{1}{R} \Delta^2 & 0 \\ -U' \partial_z & -U \partial_x + \frac{1}{R} \Delta \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix} + \begin{bmatrix} -\partial_{xy} & \partial_x^2 + \partial_z^2 & -\partial_{zy} \\ \partial_z & 0 & -\partial_x \end{bmatrix} \begin{bmatrix} d_x \\ d_y \\ d_z \end{bmatrix}$$

$$\begin{bmatrix} \tilde{u} \\ \tilde{v} \\ \tilde{w} \end{bmatrix} = (\partial_x^2 + \partial_z^2)^{-1} \begin{bmatrix} \partial_{xy} & -\partial_z \\ \partial_x^2 + \partial_z^2 & 0 \\ \partial_{zy} & \partial_x \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix}$$

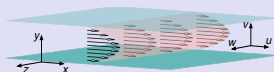


$$\begin{aligned} \partial_t \Psi &= \mathcal{A} \Psi + \mathcal{B} \mathbf{d} \\ \tilde{\mathbf{u}} &= \mathcal{C} \Psi \end{aligned}$$

Input-Output Analysis of the Linearized NS Equations

$$\partial_t \begin{bmatrix} \Delta \tilde{v} \\ \tilde{\omega} \end{bmatrix} = \begin{bmatrix} U'' \partial_x - U \Delta \partial_x + \frac{1}{R} \Delta^2 & 0 \\ -U' \partial_z & -U \partial_x + \frac{1}{R} \Delta \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix} + \begin{bmatrix} -\partial_{xy} & \partial_x^2 + \partial_z^2 & -\partial_{zy} \\ \partial_z & 0 & -\partial_x \end{bmatrix} \begin{bmatrix} d_x \\ d_y \\ d_z \end{bmatrix}$$

$$\begin{bmatrix} \tilde{u} \\ \tilde{v} \\ \tilde{w} \end{bmatrix} = (\partial_x^2 + \partial_z^2)^{-1} \begin{bmatrix} \partial_{xy} & -\partial_z \\ \partial_x^2 + \partial_z^2 & 0 \\ \partial_{zy} & \partial_x \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix}$$



$$\begin{aligned} \partial_t \Psi &= \mathcal{A} \Psi + \mathcal{B} \mathbf{d} \\ \tilde{\mathbf{u}} &= \mathcal{C} \Psi \end{aligned}$$

- eigs (\mathcal{A}): determine stability

(standard technique in *Linear Hydrodynamic Stability*)

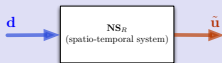
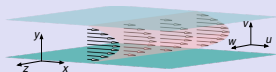
- Transfer Function $\mathbf{d} \rightarrow \tilde{\mathbf{u}}$: determines response to disturbances

(uncommon in Fluid Mechanics
an "open system")

Input-Output Analysis of the Linearized NS Equations

$$\partial_t \begin{bmatrix} \Delta \tilde{v} \\ \tilde{\omega} \end{bmatrix} = \begin{bmatrix} U'' \partial_x - U \Delta \partial_x + \frac{1}{R} \Delta^2 & 0 \\ -U' \partial_z & -U \partial_x + \frac{1}{R} \Delta \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix} + \begin{bmatrix} -\partial_{xy} & \partial_x^2 + \partial_z^2 & -\partial_{zy} \\ \partial_z & 0 & -\partial_x \end{bmatrix} \begin{bmatrix} d_x \\ d_y \\ d_z \end{bmatrix}$$

$$\begin{bmatrix} \tilde{u} \\ \tilde{v} \\ \tilde{w} \end{bmatrix} = (\partial_x^2 + \partial_z^2)^{-1} \begin{bmatrix} \partial_{xy} & -\partial_z \\ \partial_x^2 + \partial_z^2 & 0 \\ \partial_{zy} & \partial_x \end{bmatrix} \begin{bmatrix} \tilde{v} \\ \tilde{\omega} \end{bmatrix}$$



$$\begin{aligned} \partial_t \Psi &= \mathcal{A} \Psi + \mathcal{B} \mathbf{d} \\ \tilde{\mathbf{u}} &= \mathcal{C} \Psi \end{aligned}$$

Surprises:

- Even when \mathcal{A} is stable

the gain $\mathbf{d} \rightarrow \tilde{\mathbf{u}}$ can be very large
 ($(H^2 \text{ norm})^2$ scales with R^3)

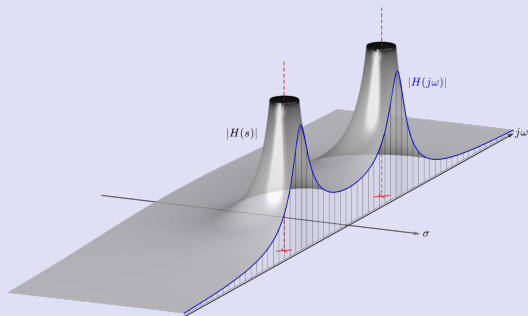
- Input-output resonances

very different from least-damped modes of \mathcal{A}

Modal vs. Input-Output Response

Typically: underdamped poles \longleftrightarrow frequency response peaks

cf. The “rubber sheet analogy”:



Modal vs. Input-Output Response

However: Pole Locations \leftrightarrow Frequency Response Peaks

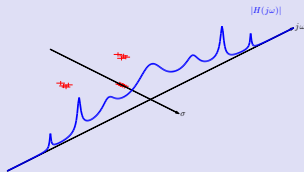
Theorem: Given any desired *pole locations*

$$z_1, \dots, z_n \in \mathbb{C}_- \text{ (LHP),}$$

and any *stable frequency response* $H(j\omega)$, arbitrarily close approximation is achievable with

$$\left\| H(s) - \left(\sum_{i=1}^{N_1} \frac{\alpha_{1,i}}{(s - z_1)^i} + \dots + \sum_{i=1}^{N_n} \frac{\alpha_{n,i}}{(s - z_n)^i} \right) \right\|_{\mathcal{H}^2} \leq \epsilon$$

by choosing any of the N_k 's large enough



Modal vs. Input-Output Response

However: Pole Locations \leftrightarrow Frequency Response Peaks

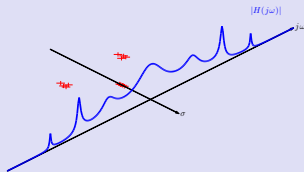
Theorem: Given any desired *pole locations*

$$z_1, \dots, z_n \in \mathbb{C}_- \text{ (LHP),}$$

and any *stable frequency response* $H(j\omega)$, arbitrarily close approximation is achievable with

$$\left\| H(s) - \left(\sum_{i=1}^{N_1} \frac{\alpha_{1,i}}{(s - z_1)^i} + \dots + \sum_{i=1}^{N_n} \frac{\alpha_{n,i}}{(s - z_n)^i} \right) \right\|_{\mathcal{H}^2} \leq \epsilon$$

by choosing any of the N_k 's large enough



Remarks:

- No necessary relation between *pole locations* and *system resonances*
- $(\epsilon \rightarrow 0 \Rightarrow N_k \rightarrow \infty)$, i.e. this is a *large-scale systems* phenomenon
- **Large-scale systems:** IO behavior not always predictable from modal behavior

Modal vs. Input-Output Response

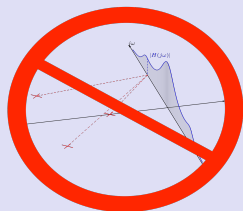
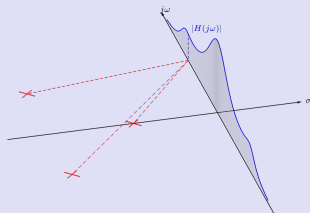
However: Pole Locations \leftrightarrow Frequency Response Peaks

MIMO case: $H(s) = (sI - A)^{-1}$

- If A is *normal* (has orthogonal eigenvectors), then

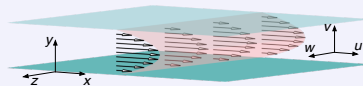
$$\sigma_{\max} \left((j\omega I - A)^{-1} \right) = \frac{1}{\text{distance}(j\omega, \text{nearest pole})}$$

- If A is *non-normal*: no clear relation between
singular value plot \leftrightarrow eigs(A)



Translation invariance in x & z implies

- *Impulse Response* (Green's Function)



$$\tilde{\mathbf{u}}(t, x, y, z) = \int G(t - \tau, x - \xi, \mathbf{y}, \mathbf{y}', z - \zeta) \mathbf{d}(\tau, \xi, \mathbf{y}', \zeta) d\tau d\xi dy' d\zeta$$

$$\tilde{\mathbf{u}}(t, x, \cdot, z) = \int \mathcal{G}(t - \tau, x - \xi, z - \zeta) \mathbf{d}(\tau, \xi, \cdot, \zeta) d\tau d\xi d\zeta$$

$\mathcal{G}(t, x, z)$: Operator-valued impulse response

- *Frequency Response*

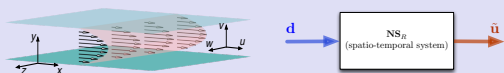
$$\tilde{\mathbf{u}}(\omega, k_x, k_z) = \mathcal{G}(\omega, k_x, k_z) \mathbf{d}(\omega, k_x, k_z)$$

$\mathcal{G}(\omega, k_x, k_z)$: Operator-valued frequency response (Packs lots of information!)

- *Spectrum of \mathcal{A} :*

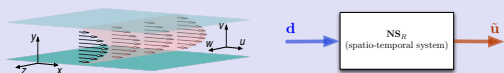
$$\sigma(\mathcal{A}) = \overline{\bigcup_{k_x, k_z} \sigma(\hat{\mathcal{A}}(k_x, k_z))}$$

Modal vs. Input-Output Analysis



- $\partial_t \Psi = \mathcal{A} \Psi + \mathcal{B} \mathbf{d}$
 $\tilde{\mathbf{u}} = \mathcal{C} \Psi$
- IR: $\mathcal{G}(t, x, z)$
- FR: $\mathcal{G}(\omega, k_x, k_z)$

Modal vs. Input-Output Analysis

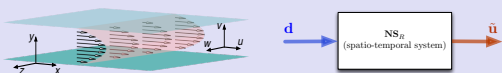


- $\partial_t \Psi = \mathcal{A} \Psi + \mathcal{B} \mathbf{d}$
- $\tilde{\mathbf{u}} = \mathcal{C} \Psi$
- IR: $\mathcal{G}(t, x, z)$
- FR: $\mathcal{G}(\omega, k_x, k_z)$

Modal Analysis: Look for unstable eigs of \mathcal{A}

Flow type	Classical linear theory R_c	Experimental R_c
Channel Flow	5772	$\approx 1,000-2,000$
Plane Couette	∞	≈ 350
Pipe Flow	∞	$\approx 2,200-100,000$

Modal vs. Input-Output Analysis

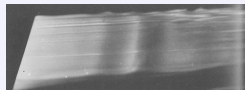
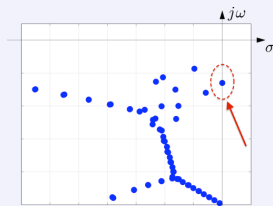


- $\partial_t \Psi = \mathcal{A} \Psi + \mathcal{B} \mathbf{d}$
- $\tilde{\mathbf{u}} = \mathcal{C} \Psi$
- IR: $\mathcal{G}(t, x, z)$
- FR: $\mathcal{G}(\omega, k_x, k_z)$

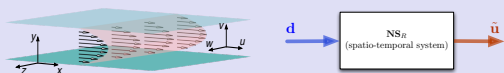
Modal Analysis: Look for unstable eigs of \mathcal{A}

- Channel Flow @ $R = 6000, k_x = 1, k_z = 0$:

- Flow structure of corresponding eigenfunction:
Tollmein-Schlichting (TS) waves

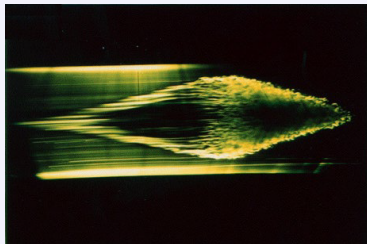
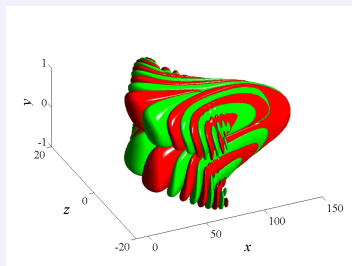


Modal vs. Input-Output Analysis



- $\partial_t \Psi = \mathcal{A} \Psi + \mathcal{B} \mathbf{d}$
- $\tilde{\mathbf{u}} = \mathcal{C} \Psi$
- IR: $G(t, x, y, -1, z)$
- FR: $\mathcal{G}(\omega, k_x, k_z)$

Impulse Response Analysis: Channel Flow @ $R = 2000$

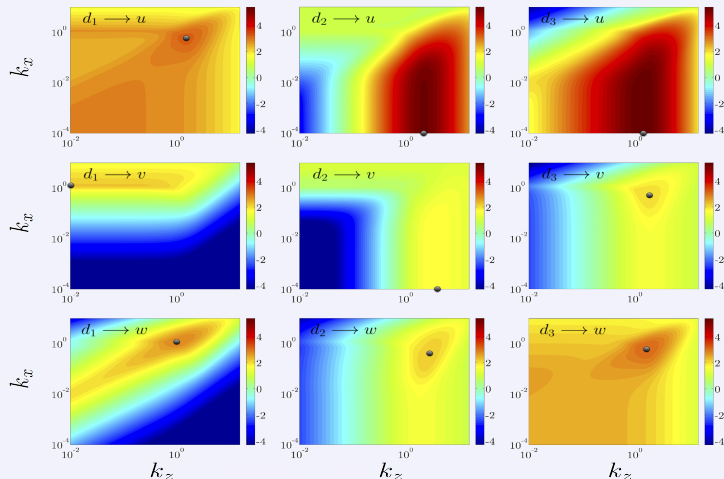


similar to “turbulent spots”

Spatio-temporal Frequency Response

$\mathcal{G}(\omega, k_x, k_z)$ is a *large* object!

one aggregation method: $\sup_{\omega} \sigma_{\max} \left(\mathcal{G}(\omega, k_x, k_z) \right)$

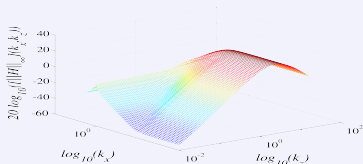
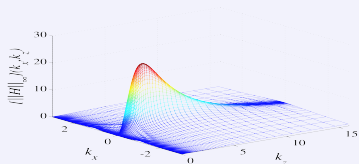


Jovanovic, BB, '05 JFM

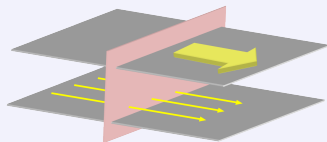
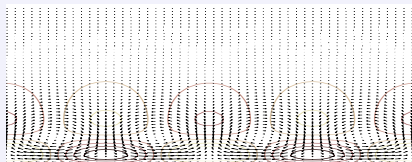
Spatio-temporal Frequency Response

$\mathcal{G}(\omega, k_x, k_z)$ is a *large* object!

one aggregation method: $\sup_{\omega} \sigma_{\max} \left(\mathcal{G}(\omega, k_x, k_z) \right)$



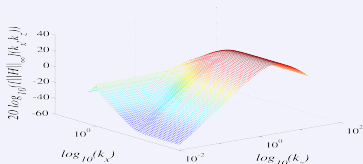
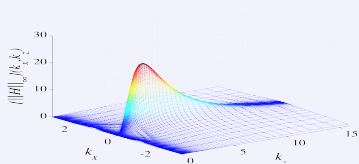
What do the corresponding flow structures look like?



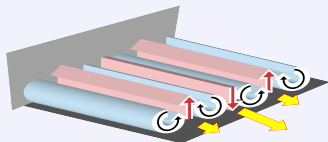
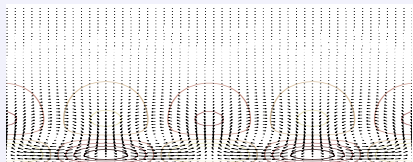
Spatio-temporal Frequency Response

$\mathcal{G}(\omega, k_x, k_z)$ is a *large* object!

one aggregation method: $\sup_{\omega} \sigma_{\max} \left(\mathcal{G}(\omega, k_x, k_z) \right)$



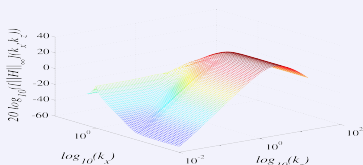
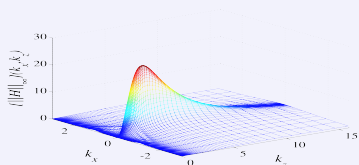
What do the corresponding flow structures look like?



Spatio-temporal Frequency Response

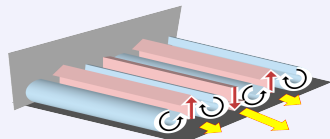
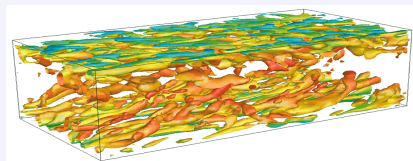
$\mathcal{G}(\omega, k_x, k_z)$ is a *large* object!

one aggregation method: $\sup_{\omega} \sigma_{\max} \left(\mathcal{G}(\omega, k_x, k_z) \right)$



What do the corresponding flow structures look like?

closer (than TS waves) to structures seen in turbulent boundary layers



Flow Control

Some recent related progress in Fluid Dynamics and Controls communities

- Henningson & Co. @ KTH
- Rowley & Co. @ Princeton
- Gayme, Doyle & Mckeon @ Caltech
- Jovanovic & Co. @ Minnesota

Viscoelastic turbulence

Vibrational Control with Wall Oscillations

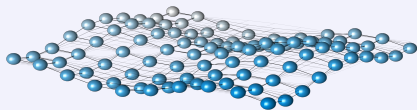
FLOW CONTROL remains

- an *under-explored* field
- with many *high-payoff* possibilities
(both intellectually and technologically)

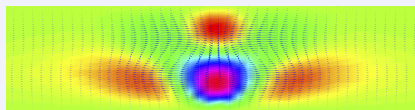
Recap

SPATIALLY DISTRIBUTED SYSTEMS

Networked/Cooperative/Distributed Control



Distributed Parameter Systems



SOME COMMON THEMES EMERGE

- *The use of system norms and responses*
- *Large-scale & Regular Networks* → *Asymptotic statements (in system size)*
- *Network topology imposes asymptotic “hard performance limits”*
- *Large-scale (even linear) systems exhibit some surprising phenomena*
- This is a very rich area with many remaining
 - fascinating questions, unsolved problems
 - research problems yet to be properly formulated

Collaborators

- M. Jovanovic
- D. Gayme
- S. Patterson
- J.C. Doyle
- B. Mckeon

- M. Dahleh
- P. Mitra
- P. Voulgaris
- F. Paganini
- M.A. Dahleh

Support:



Energy, Power & Adaptive Systems Program (ECCS)

Control Systems (CMMI)

Physics of Living Systems (PHY)



Dynamics & Control Program